## The Finite Element Method for the Analysis of Non-Linear and Dynamic Systems



Prof. Dr. Michael Havbro Faber
Swiss Federal Institute of Technology ETH Zurich, Switzerland


## Contents of Today's Lecture

- Short summary of the main findings from the last lecture
- Aim of the present lecture - in short : $^{\text {- }}$
- The deformation gradient, strain and stress tensors
- Continuum mechanics formulations
- incremental total Lagrangian
- incremental updated Lagrangian
- materially non-linear analysis only


## Short summary of the last lecture

- The basic approach in incremental anaylsis is
${ }^{t+\Delta t} \mathbf{R}-{ }^{t+\Delta t} \mathbf{F}=0$
assuming that ${ }^{t+\Delta t} \mathbf{R}$ is independent of the deformations we have
${ }^{t+\Delta t} \mathbf{F}={ }^{t} \mathbf{F}+\mathbf{F}$
We know the solution ${ }^{t} F$ at time $t$ and $F$ is the increment in the nodal point forces corresponding to an increment in the displacements and stresses from time $t$ to time $t+\Delta t$. This we can approximate by:
$\mathbf{F}={ }^{t} \mathbf{K} \mathbf{U}$
1
Tangent stiffness matrix $\quad{ }^{t} \mathbf{K}=\frac{\partial^{t} \mathbf{F}}{\partial^{t} \mathbf{U}}$


## Short summary of the last lecture

- The basic approach in incremental anaylsis is

We may now substitute the tangent stiffness matrix into the equlibrium relation

```
't}\mathbf{KU}=\mp@subsup{}{}{t+\Deltat}\mathbf{R}-\mp@subsup{}{}{t}\mathbf{F
\Downarrow
*+\Deltat}\mathbf{U}=\mp@subsup{}{}{t}\mathbf{U}+\mathbf{U
```

which gives us a scheme for the calculation of the displacements

The exact displacements at time $t+\Delta t$ correspond to the applied loads at $t+\Delta t$, however, we only determined these approximately as we used a tangent stiffness matrix - thus we may have to iterate to find the solution


## Short summary of the last lecture

- The basic approach in incremental anaylsis is

We may use the Newton-Raphson iteration scheme to find the equlibrium within each load increment:

$$
\begin{aligned}
& { }^{t+\Delta t} \mathbf{K}^{(i-1)} \Delta \mathbf{U}^{(i)}={ }^{t+\Delta t} \mathbf{R}-{ }^{t+\Delta t} \mathbf{F}^{(i-1)} \quad \text { (out of balance load vector) } \\
& { }^{t+\Delta t} \mathbf{U}^{(i)}={ }^{t+\Delta t} \mathbf{U}^{(i-1)}+\Delta \mathbf{U}^{(i)} \\
& \text { with initial conditions } \\
& { }^{t+\Delta t} \mathbf{U}^{(0)}={ }^{t} \mathbf{U} ; \quad{ }^{t+\Delta t} \mathbf{K}^{(0)}={ }^{t} \mathbf{K} ; \quad{ }^{t+\Delta t} \mathbf{F}^{(0)}={ }^{t} \mathbf{F}
\end{aligned}
$$



## Short summary of the last lecture

- The basic approach in incremental anaylsis is

It may be expensive to calculate the tangent stiffness matrix and;

In the Modified Newton-Raphson iteration scheme it is thus only calculated in the beginning of each new load step

In the Quasi-Newton iteration schemes the secant stiffness matrix is used instead of the tangent matrix

## Short summary of the last lecture

- The basic problem:

We want to establish the solution to a non-linear mechanical problem using an incremental formulation

The equilibrium must be established for the considered body in its current configuration

In proceeding we adopt a Lagrangian formulation where we track the movement of all particles of the body (located in a Cartesian coordinate system)

Another approach would be an Eulerian formulation where the motion of material through a stationary control volume is considered

## Short summary of the last lecture

- The basic problem:



## Short summary of the last lecture

- The Lagrangian formulation

We express equilibrium of the body at time $\boldsymbol{t + \Delta t}$ using the principle of virtual displacements
$\int_{t+\Delta_{V}}{ }^{t+\Delta t} \tau \delta_{t+\Delta t} e_{i j} d^{t+\Delta t} V={ }^{t+\Delta t} R$

${ }^{t+\Delta t} \tau$ : Cartesian components of the Cauchy stress tensor
$\delta_{t+\Delta t} e_{i j}=\frac{1}{2}\left(\frac{\partial \delta u_{i}}{\partial^{t+\Delta t} x_{j}}+\frac{\partial \delta u_{j}}{\partial^{t+\Delta t} x_{i}}\right)=$ strain tensor corresponding to virtual displacements
$\delta u_{i}$ : Components of virtual displacement vector imposed at time $t+\Delta t$
${ }^{t+\Delta t} x_{i}$ : Cartesian coordinate at time $t+\Delta t$
${ }^{t+\Delta t} V$ : Volume at time $t+\Delta t$
${ }^{t+\Delta t} R=\int_{t+\Delta^{t} V}{ }^{t+\Delta t} f_{i}^{B} \delta u_{i} d^{t+\Delta t} V+\int_{t+\Delta S_{S}}{ }^{t+\Delta t} f_{i}^{S} \delta u_{i}^{S} d^{t+\Delta t} S$

## Short summary of the last lecture

- The Lagrangian formulation

We express equilibrium of the body at time $\boldsymbol{t + \Delta t}$ using the principle of virtual displacements


where
${ }^{t+\Delta t} f_{i}^{B}$ : externally applied forces per unit volume
${ }^{t+\Delta t} f_{i}^{S}$ : externally applied surface tractions per unit surface
${ }^{t+\Delta t} S_{f}$ : surface at time $t+\Delta t$
$\delta u_{i}^{S}: \delta u_{i}$ evaluated at the surface ${ }^{t+\Delta t} S_{f}$

## Short summary of the last lecture

- The Lagrangian formulation

We recognize that our derivations from linear finite element theory are unchanged - but applied to the body in the configuration at time $\boldsymbol{t}+\Delta \boldsymbol{t}$

## Short summary of the last lecture

- In the further we introduce an appropriate notation:

Coordinates and displacements are related as:

$$
\begin{aligned}
& { }^{t} x_{i}={ }^{0} x_{i}+{ }^{t} u_{i} \\
& { }^{t+\Delta t} x_{i}={ }^{0} x_{i}+{ }^{t+\Delta \Delta} u_{i}
\end{aligned}
$$

Increments in displacements are related as:

$$
{ }_{t} u_{i}={ }^{t+\Delta t} u_{i}-{ }^{t} u_{i}
$$

Reference configurations are indexed as e.g.:
${ }^{t+\Delta t}{ }_{0}^{S}{ }_{i}^{S}$ where the lower left index indicates the reference configuration

$$
{ }^{t+\Delta t} \tau_{i j}{ }{ }^{t+\Delta t}+\frac{t \Delta t}{}
$$

Differentiation is indexed as:

$$
{ }_{0}^{t+\Delta t} u_{i, j}=\frac{\partial^{t+\Delta t} u_{i}}{\partial^{0} x_{j}}, \quad{ }_{t+\Delta t}^{0} x_{m, n}=\frac{\partial^{0} x_{m}}{\partial^{t+\Delta t} x_{n}}
$$

## Aim of the present lecture

- We have already formulated the continuum mechanich incremental equations of motion
$\int_{t+\Delta \Delta_{V}}{ }^{t+\Delta t} \tau \delta_{t+\Delta t} e_{i j} d^{t+\Delta t} V={ }^{t+\Delta t} R$
and
${ }^{t+\Delta t} R=\int_{t+\Delta V_{V}}{ }^{t+\Delta t} f_{i}^{B} \delta u_{i} d^{t+\Delta t} V+\int_{t+\Delta \Delta S_{f}}{ }^{t+\Delta t} f_{i}^{S} \delta u_{i}^{S} d^{t+\Delta t} S$
a basic problem is that we don't know the configuration at time $t+\Delta t$ (in linear analysis we always used the original configuration as basis)
what we need to do now is to introduce appropriate stress and strain measures as well as constitutive relations



## The deformation gradient, strain and stress tensors

- As mentioned - we must try to establish a description of the volume we consider such that we can express the internal virtual work in terms of an integral over a volume we know!
- Further we would like to be able to decompose the stresses and strains in an efficient manner - keeping track of how the volume stretches and how it rotates (rigidly).



## The deformation gradient, strain and stress tensors

We consider a body under deformation at times 0 and $\boldsymbol{t}$


## The deformation gradient, strain and stress tensors

We now consider the change of an infinitesimal gradient vector


We define

$$
d^{t} \mathbf{x}={ }_{0}^{t} \mathbf{X} d^{0} \mathbf{x}
$$



## The deformation gradient, strain and stress tensors

- We can write the deformation gradient as

$$
{ }_{0}^{t} \mathbf{X}=\left[\begin{array}{lll}
\frac{\partial^{t} x_{1}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{1}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{1}}{\partial^{0} x_{3}} \\
\frac{\partial^{\prime} x_{2}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{2}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{2}}{\partial^{0} x_{3}} \\
\frac{\partial^{t} x_{3}}{\partial^{0} x_{1}} & \frac{\partial^{\prime} x_{3}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{3}}{\partial^{0} x_{3}}
\end{array}\right]
$$

The deformation gradient describes the stretches and rotations that the material fibers have undergone from time zero to time $t$
${ }_{0}^{t} \mathbf{X}=\left({ }_{0} \nabla^{t} \mathbf{x}^{T}\right)^{T}$, where $\quad{ }_{0} \nabla=\left[\begin{array}{c}\frac{\partial}{\partial^{0} x_{1}} \\ \frac{\partial}{\partial^{0} x_{2}} \\ \frac{\partial}{\partial^{0} x_{3}}\end{array}\right] ; \quad$ and $\quad{ }^{t} \mathbf{x}^{T}=\left[\begin{array}{lll}{ }^{t} x_{1} & { }^{t} x_{2} & { }^{t} x_{3}\end{array}\right]$
it can be show that ${ }_{0}^{t} \mathbf{X}=\left({ }_{t}^{0} \mathbf{X}\right)^{-1}$ and $\quad{ }^{t} \rho=\frac{{ }^{0} \rho}{\operatorname{det}\left({ }_{t}^{0} \mathbf{X}\right)}$

## The deformation gradient, strain and stress tensors

- Then we introduce the Cauchy-Green deformation tensor

The deformation gradient is also used to measure the stretch of a material fiber and the change in angle between fibers due to the deformation

$$
\begin{aligned}
& { }_{0}^{t} \mathbf{C}={ }_{0}^{t} \mathbf{X}^{T}{ }_{0}^{t} \mathbf{X} \quad \begin{array}{l}
\text { "Right Cauchy-Green } \\
\text { deformation tensor" }
\end{array} \\
& { }_{0}^{t} \mathbf{B}={ }_{0}^{t} \mathbf{X}_{0}^{t} \mathbf{X}^{T} \quad \begin{array}{l}
\text { "Left Cauchy-Green } \\
\text { deformation tensor"" }
\end{array}
\end{aligned}
$$

${ }_{0}^{t} \mathbf{X}=\left[\begin{array}{lll}\frac{\partial^{t} x_{1}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{1}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{1}}{\partial^{0} x_{3}} \\ \frac{\partial^{t} x_{2}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{2}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{2}}{\partial^{0} x_{3}} \\ \frac{\partial^{t} x_{3}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{3}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{3}}{\partial^{0} x_{3}}\end{array}\right]$

$$
{ }_{0}^{{ }_{0}^{t} \mathbf{X}=\left(\begin{array}{llll}
\left.{ }_{0} \nabla^{t} \mathbf{x}^{T}\right)^{T}, & \text { where } & { }_{0} \nabla=\left[\begin{array}{lll}
0 & x_{1} & O \\
x_{2} & & 0
\end{array} x_{3}\right.
\end{array}\right]\left[\begin{array}{l}
\frac{\partial}{\partial^{0} x_{1}} \\
\frac{\partial}{\partial^{0} x_{2}} \\
\frac{\partial}{\partial^{0} x_{3}}
\end{array}\right] ; \quad \text { and } \quad{ }^{t} \mathbf{x}^{T}=\left[\begin{array}{lll}
{ }^{t} x_{1} & { }^{t} x_{2}
\end{array}\right]}
$$



## The deformation gradient, strain and stress tensors

- The deformation gradient

The deformation gradient can be decomposed into a unique product of two matrices
${ }_{0}^{t} \mathbf{X}={ }_{0}^{t} \mathbf{R}{ }_{0}^{t} \mathbf{U}$
${ }_{0}^{t} \mathbf{U} \quad$ Symmetric stretch matrix


${ }_{0}^{t} \mathbf{R} \quad$ Orthogonal rotation matrix

Referred to as a polar decomposition (illustrated in Ex 6.8)
Sometimes the indexes referring to time are omitted!

## The deformation gradient, strain and stress tensors

- Decomposition of the deformation gradient

We continue by rewriting the deformation gradient
$\mathbf{X}=\mathbf{R U}=\mathbf{R U R}^{T} \mathbf{R}=\mathbf{V R}$
${ }_{0}^{t} \mathbf{X}={ }_{0}^{t} \mathbf{R}_{0}^{t} \mathbf{U}$
${ }_{0}^{t} \mathbf{U} \quad$ Symmetric stretch matrix
${ }_{0}^{t} \mathbf{R} \quad$ Orthogonal rotation matrix

U: right stretch matrix
V: left stretch matrix
Further it can be shown (Ex 6.8) that :
$\mathbf{U}=\mathbf{R}_{L} \mathbf{\Lambda} \mathbf{R}_{L}^{T}$
A: Principal stretches
$\mathbf{R}_{L}$ : Direction of principal stretches

## The deformation gradient, strain and stress tensors

- Decomposition of the deformation gradient

There is also:

| $\mathbf{U}=\mathbf{R}_{L} \boldsymbol{\Lambda} \mathbf{R}_{L}^{T}$ |
| :--- |
| $\boldsymbol{\Lambda}: \quad$ Principal stretches |
| $\mathbf{R}_{L}:$ Direction of principal stretches |

$\mathbf{V}=\mathbf{R}_{E} \mathbf{\Lambda} \mathbf{R}_{E}^{T}$
$\mathbf{R}_{E}$ : Base vectors of principal stretches
in the stationary coordinate system

## The deformation gradient, strain and stress tensors

We consider a bar under stretch and rotation


## The deformation gradient, strain and stress tensors

- Using the decomposition of the deformation gradient we may rewrite the right and left Cauchy-Green deformation tensors:

The right Cauchy-Green deformation tensor:
$\mathbf{C}=\mathbf{X}^{T} \mathbf{X}=\mathbf{U R}^{T} \mathbf{R U}=\mathbf{U}^{2}$

The left Cauchy-Green deformation tensor:

$$
\mathbf{B}=\mathbf{X X}^{T}=\mathbf{V R R}^{T} \mathbf{V}=\mathbf{V}^{2}
$$

## The deformation gradient, strain and stress tensors

- We now proceed from deformations to strains ()

The strain may be understood as the stretch per unit length why we can assess the strain through the inner product between two infinitesimal vectors before and after deformation

$$
\begin{aligned}
& d^{t} \mathbf{x}_{1} \cdot d^{t} \mathbf{x}_{2}-d^{0} \mathbf{x}_{1} \bullet d^{0} \mathbf{x}_{2}=\left(\mathbf{X} d^{0} \mathbf{x}_{1}\right) \cdot\left(\mathbf{X} d^{0} \mathbf{x}_{2}\right)-d^{0} \mathbf{x}_{1} \cdot d^{0} \mathbf{x}_{2} \\
& =d^{0} \mathbf{x}_{1} \bullet(\mathbf{C}-\mathbf{I}) \bullet d^{0} \mathbf{x}_{2} \\
& \text { Green-Lagrange strain: } \frac{1}{2}(\mathbf{C}-\mathbf{I}) \\
& d^{t} \mathbf{x}_{1} \bullet d^{t} \mathbf{x}_{2}-d^{0} \mathbf{x}_{1} \cdot d^{0} \mathbf{x}_{2}=d^{t} \mathbf{x}_{1} \bullet d^{t} \mathbf{x}_{2}-\left(\mathbf{X}^{-1} d^{t} \mathbf{x}_{1}\right) \bullet\left(\mathbf{X}^{-1} d^{t} \mathbf{x}_{2}\right) \\
& =d^{t} \mathbf{x}_{1} \bullet\left(\mathbf{I}-\mathbf{B}^{-1}\right) \cdot d^{t} \mathbf{x}_{2} \\
& \text { Almansi strain: } \frac{1}{2}\left(\mathbf{I}-\mathbf{B}^{-1}\right)
\end{aligned}
$$



## The deformation gradient, strain and stress tensors

- Lets see an example (one-dimensional) We assume the following deformation gradient matrix

$$
\begin{aligned}
\mathbf{X}=\left[\begin{array}{ccc}
\frac{l}{L} & 0 & 0 \\
0 & 1 & 0 \\
0 & 0 & 1
\end{array}\right] ; \quad \text { i.e. pure stretch } \\
\begin{aligned}
d^{t} \mathbf{x}_{1} \cdot d^{t} \mathbf{x}_{2}-d^{0} \mathbf{x}_{1} \bullet d^{0} \mathbf{x}_{2} & =\left(\frac{l}{L} d^{0} \mathbf{x}_{1}\right) \cdot\left(\frac{l}{L} d^{0} \mathbf{x}_{2}\right)-d^{0} \mathbf{x}_{1} \bullet d^{0} \mathbf{x}_{2} \\
& =d^{0} \mathbf{x}_{1} \bullet\left(\frac{l^{2}}{L^{2}}-1\right) \cdot d^{0} \mathbf{x}_{2} \\
& \text { or equivalently } \\
& =d^{t} \mathbf{x}_{1} \bullet d^{t} \mathbf{x}_{2}-\left(\mathbf{X}^{-1} d^{t} \mathbf{x}_{1}\right) \bullet\left(\mathbf{X}^{-1} d^{t} \mathbf{x}_{2}\right) \\
& =d^{t} \mathbf{x}_{1} \bullet\left(1-\frac{L^{2}}{l^{2}}\right) \cdot d^{t} \mathbf{x}_{2}
\end{aligned}
\end{aligned}
$$

## The deformation gradient, strain and stress tensors

- Lets see an example (one-dimensional)

Green-Lagrange strains: $\quad \mathbf{E}=\frac{1}{2}\left(\frac{l^{2}}{L^{2}}-1\right)$
Almansi strains: $\quad \mathbf{A}=\frac{1}{2}\left(1-\frac{L^{2}}{l^{2}}\right)$
for infinitesimal strains there is: $\frac{1}{2}\left(\frac{l^{2}}{L^{2}}-1\right)=\frac{1}{2} \frac{(u+L)^{2}-L^{2}}{L^{2}} \approx \frac{u}{L}$ and

$$
\frac{1}{2}\left(1-\frac{L^{2}}{l^{2}}\right)=\frac{1}{2} \frac{(u+L)^{2}-L^{2}}{l^{2}} \approx \frac{u}{l} \approx \frac{u}{L}
$$

## The deformation gradient, strain and stress tensors

- We now consider the tensor components of the strain tensors


## Green-Lagrange strains

$\boldsymbol{\varepsilon}=\varepsilon_{i j} \mathbf{e}_{i} \otimes \mathbf{e}_{j}=\frac{1}{2}\left\{\frac{\partial u_{i}}{\partial^{o} x_{j}}+\frac{\partial u_{j}}{\partial^{o} x_{i}}+\frac{\partial u_{k}}{\partial^{o} x_{i}} \frac{\partial u_{k}}{\partial^{o} x_{j}}\right\} \mathbf{e}_{i} \otimes \mathbf{e}_{j}$
Almansi strains

$$
\boldsymbol{\alpha}=\alpha_{i j} \mathbf{e}_{i} \otimes \mathbf{e}_{j}=\frac{1}{2}\left\{\frac{\partial u_{i}}{\partial^{t} x_{j}}+\frac{\partial u_{j}}{\partial^{t} x_{i}}+\frac{\partial u_{k}}{\partial^{t} x_{i}} \frac{\partial u_{k}}{\partial^{t} x_{j}}\right\} \mathbf{e}_{i} \otimes \mathbf{e}_{j}
$$

## The deformation gradient, strain and stress tensors

$$
\begin{aligned}
& \text { Example - beam element } \\
& \mathbf{X}=\mathbf{R U} \\
& \mathbf{U}=\left[\begin{array}{ccc}
\frac{l}{L} & 0 & 0 \\
0 & \frac{h}{H} & 0 \\
0 & 0 & \frac{h}{H}
\end{array}\right] ; \quad \mathbf{R}=\left[\begin{array}{ccc}
\cos \theta & -\sin \theta & 0 \\
\sin \theta & \cos \theta & 0 \\
0 & 0 & 1
\end{array}\right] ; \quad \mathbf{X}=\left[\begin{array}{c}
t \\
t_{2}
\end{array}\right]
\end{aligned}
$$

## The deformation gradient, strain and stress tensors

Now we consider the velocity gradient tensor - the difference in velocity of two points infinitesimally close


We can write change of velocity over space as a linear function of the distance in space
$d^{t} \mathbf{v}=\mathbf{L} d^{t} \mathbf{x}$
where $L$ is given through the gradient of the velocity field at time $t$
$\mathbf{L}=\mathbf{v} \otimes \nabla_{\mathbf{x}} \quad$ This is the velocity gradient tensor ©


## The deformation gradient, strain and stress tensors

We remember that there is:
$d^{t} \mathbf{x}=\mathbf{X} d^{0} \mathbf{x}$
which leads us to:


$$
\begin{aligned}
& d^{t} \mathbf{v}=\dot{\mathbf{X}} d^{0} \mathbf{x} \\
& d^{t} \mathbf{v}=\mathbf{L} \mathbf{X} d^{0} \mathbf{x} \\
& \Downarrow \\
& \mathbf{L}=\dot{\mathbf{X}} \mathbf{X}^{-1}
\end{aligned}
$$

$\mathbf{L}=\mathbf{D}+\mathbf{W}$ decomposition
$\mathbf{D}=\frac{1}{2}\left(\mathbf{L}+\mathbf{L}^{T}\right)=\frac{1}{2}\left(\frac{\partial v_{i}}{\partial x_{j}}+\frac{\partial v_{j}}{\partial x_{i}}\right) \mathbf{e}_{i} \otimes \mathbf{e}_{j}$
deformation rate tensor
$\mathbf{W}=\frac{1}{2}\left(\mathbf{L}-\mathbf{L}^{T}\right)=\frac{1}{2}\left(\frac{\partial v_{i}}{\partial x_{j}}-\frac{\partial v_{j}}{\partial x_{i}}\right) \mathbf{e}_{i} \otimes \mathbf{e}_{j}$
spin/rotation rate tensor

## The deformation gradient, strain and stress tensors

And then we may derive the Green-Lagrange velocity strain tensor

$$
\dot{\boldsymbol{\varepsilon}}={ }_{0}^{t} \mathbf{X}^{T} \mathbf{D}_{0}^{t} \mathbf{X} \quad \mathbf{D}={ }_{t}^{0} \mathbf{X}^{T} \dot{\boldsymbol{\varepsilon}}_{t}^{0} \mathbf{X}
$$



We could also just have differentiated the Green-Lagrange strain tensor with resect to time

$$
\dot{\boldsymbol{\varepsilon}}=\frac{1}{2}\left({ }_{0}^{t} \dot{\mathbf{X}}^{T}{ }_{0}^{t} \mathbf{X}+{ }_{0}^{t} \mathbf{X}^{T}{ }_{0}^{t} \dot{\mathbf{X}}\right)
$$

## The deformation gradient, strain and stress tensors

Finally we need to establish the stresses
We start by introducing the Cauchy stresses:
$\mathbf{t}^{(n)}=\boldsymbol{\tau} \mathbf{n}$
$\mathbf{t}^{(n)}=\boldsymbol{\tau}_{1} \mathbf{n}_{1}+\boldsymbol{\tau}_{2} \mathbf{n}_{2}+\boldsymbol{\tau}_{3} \mathbf{n}_{3}$
$\left[\begin{array}{l}t_{1}^{(n)} \\ t_{2}^{(n)} \\ t_{3}^{(n)}\end{array}\right]=\left[\begin{array}{lll}\tau_{11} & \tau_{12} & \tau_{13} \\ \tau_{21} & \tau_{22} & \tau_{23} \\ \tau_{31} & \tau_{32} & \tau_{33}\end{array}\right]\left[\begin{array}{l}n_{1} \\ n_{2} \\ n_{3}\end{array}\right]$
Cauchy stress tensor


## The deformation gradient, strain and stress tensors

Finally we introduce the second Piola-Kirchoff stresses:
${ }_{0}^{t} \mathbf{S}=\frac{{ }^{0} \rho}{{ }^{t} \rho}{ }_{0}{ }^{0} \mathbf{X}^{t} \boldsymbol{\tau}_{t}^{0} \mathbf{X}^{T}$
these are so-called work conjugate to the Green-Lagrange strains

Rigid body motions do not induce strains/stresses
the strain and stress tensors are invariant in regard to rotations

Worthwhile to consult Ex 6.14-6.15 ©

## The deformation gradient, strain and stress tensors

## We remember that we set out to solve the following equation:

$$
\int_{t+\Delta t_{V}}{ }^{t+\Delta t} \tau \delta_{t+\Delta t} e_{i j} d^{t+\Delta t} V={ }^{t+\Delta t} R
$$

${ }^{t+\Delta t} \tau$ : Cartesian components of the Cauchy stress tensor
$\delta_{t+\Delta t} e_{i j}=\frac{1}{2}\left(\frac{\partial \delta u_{i}}{\partial^{t+\Delta t} x_{j}}+\frac{\partial \delta u_{j}}{\partial^{t+\Delta t} x_{i}}\right)=$ strain tensor corresponding to virtual displacements
$\delta u_{i}$ : Components of virtual displacement vector imposed at time $t+\Delta t$
${ }^{t+\Delta t} x_{i}$ : Cartesian coordinate at time $t+\Delta t$
${ }^{t+\Delta t} V$ : Volume at time $t+\Delta t$
${ }^{t+\Delta t} R=\int_{{ }^{t+\Delta t} V}{ }^{t+\Delta t} f_{i}^{B} \delta u_{i} d^{t+\Delta t} V+\int_{{ }^{t+\Delta t} S_{f}}{ }^{t+\Delta t} f_{i}^{S} \delta u_{i}^{S} d^{t+\Delta t} S$

## The deformation gradient, strain and stress tensors

We remember that we set out to solve the following equation:

$$
\int_{t+\Delta t}{ }^{t+\Delta t} \tau \delta_{t+\Delta t} e_{i j} d^{t+\Delta t} V={ }^{t+\Delta t} R
$$

Two schemes have been formulated for this namely:
The Total Lagrangian (TL) formulation

$$
\int_{V_{V}}{ }^{t+\Delta t} S_{i j} \delta^{t+\Delta t}{ }_{0}^{t j} \varepsilon_{i j} d^{0} V={ }^{t+\Delta t} R
$$

The Updated Lagrangian (UL) formulation

$$
\int_{V_{V}}{ }_{t}^{t+{ }_{t}} S_{i j} \delta^{t+\Delta t}{ }_{t}^{t j} \varepsilon_{i j} d^{t} V={ }^{t+\Delta t} R
$$



## The deformation gradient, strain and stress tensors

The resulting equations of motion for time $\boldsymbol{t}$ may be derived to:
The Total Lagrangian (TL) formulation
$\int_{0_{V}}{ }_{0} C_{i j r s} e_{r s} \delta_{0} e_{0} d^{0} V+\int_{\partial_{V}}{ }_{0} S_{i j} \delta_{0} \eta_{i j} d^{0} V={ }^{t+\Delta t} R-\int_{\partial_{V}}{ }_{0} S_{i j} \delta_{0} e_{i j} d^{0} V$
The Updated Lagrangian (UL) formulation
$\int_{V}{ }_{0} C_{i j r s t} e_{r s} \delta_{t} e_{i j} d^{t} V+\int_{\sigma_{V}} \tau_{i j} \delta_{t} \eta_{i j} d^{t} V={ }^{t+\Delta t} R-\int_{V}{ }_{V} \tau_{i j} \delta_{t} e_{i j} d^{t} V$
Finally - in practice it is often sufficient to account for only material non-linearity

In this case the TL and the UL formulations become identical.

