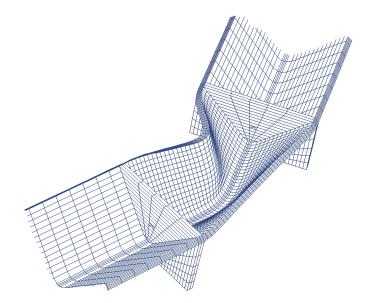


# The Finite Element Method for the Analysis of Non-Linear and Dynamic Systems



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#### **Contents of Today's Lecture**

- Short summary of the main findings from the last lecture
- Aim of the present lecture in short ③
- The deformation gradient, strain and stress tensors
- Continuum mechanics formulations
  - incremental total Lagrangian
  - incremental updated Lagrangian
  - materially non-linear analysis only

• The basic approach in incremental anaylsis is

 $^{t+\Delta t}\mathbf{R}-^{t+\Delta t}\mathbf{F}=0$ 

assuming that  ${}^{t+\Delta t}\mathbf{R}$  is independent of the deformations we have  ${}^{t+\Delta t}\mathbf{F} = {}^{t}\mathbf{F} + \mathbf{F}$ 

We know the solution <sup>t</sup>F at time t and F is the increment in the nodal point forces corresponding to an increment in the displacements and stresses from time t to time  $t+\Delta t$ . This we can approximate by:

$$\mathbf{F} = {}^{t}\mathbf{K}\mathbf{U}$$

$$\int$$
**Tangent stiffness matrix**

$${}^{t}\mathbf{K} = \frac{\partial {}^{t}\mathbf{F}}{\partial {}^{t}\mathbf{U}}$$
Method of Finite Elements II



• The basic approach in incremental anaylsis is

# We may now substitute the tangent stiffness matrix into the equibrium relation

$${}^{t}\mathbf{K}\mathbf{U} = {}^{t+\Delta t}\mathbf{R} - {}^{t}\mathbf{F}$$

$$\downarrow$$

$${}^{t+\Delta t}\mathbf{U} = {}^{t}\mathbf{U} + \mathbf{U}$$

which gives us a scheme for the calculation of the displacements

The exact displacements at time  $t+\Delta t$  correspond to the applied loads at  $t+\Delta t$ , however, we only determined these approximately as we used a tangent stiffness matrix – thus we may have to iterate to find the solution



• The basic approach in incremental anaylsis is

We may use the **Newton-Raphson** iteration scheme to find the equibrium within each load increment:

$$^{t+\Delta t}\mathbf{K}^{(i-1)}\Delta\mathbf{U}^{(i)} = {}^{t+\Delta t}\mathbf{R} - {}^{t+\Delta t}\mathbf{F}^{(i-1)}$$
 (out of balance load vector)

$$^{t+\Delta t}\mathbf{U}^{(i)} = {}^{t+\Delta t}\mathbf{U}^{(i-1)} + \Delta\mathbf{U}^{(i)}$$

with initial conditions

 $^{t+\Delta t}\mathbf{U}^{(0)} = {}^{t}\mathbf{U}; \quad {}^{t+\Delta t}\mathbf{K}^{(0)} = {}^{t}\mathbf{K}; \quad {}^{t+\Delta t}\mathbf{F}^{(0)} = {}^{t}\mathbf{F}$ 



• The basic approach in incremental anaylsis is

It may be expensive to calculate the tangent stiffness matrix and;

In the Modified Newton-Raphson iteration scheme it is thus only calculated in the beginning of each new load step

In the **Quasi-Newton** iteration schemes the secant stiffness matrix is used instead of the tangent matrix



• The basic problem:

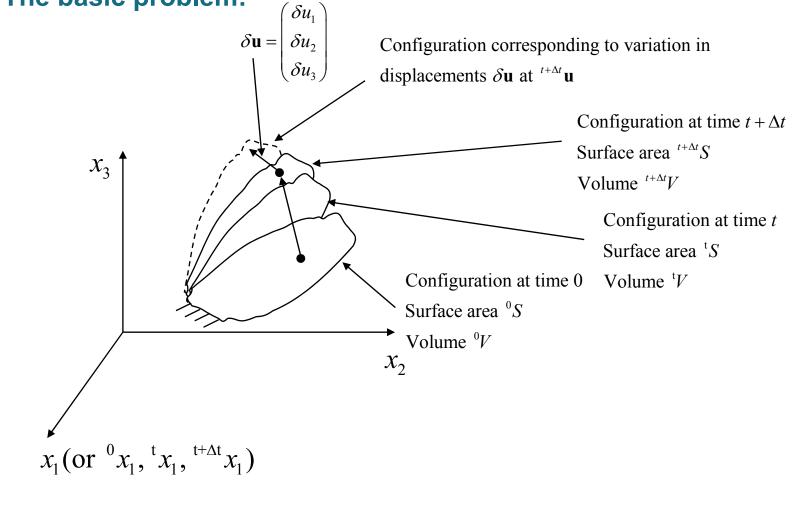
We want to establish the solution to a non-linear mechanical problem using an incremental formulation

The equilibrium must be established for the considered body in its current configuration

In proceeding we adopt a Lagrangian formulation where we track the movement of all particles of the body (located in a Cartesian coordinate system)

Another approach would be an Eulerian formulation where the motion of material through a stationary control volume is considered

#### • The basic problem:

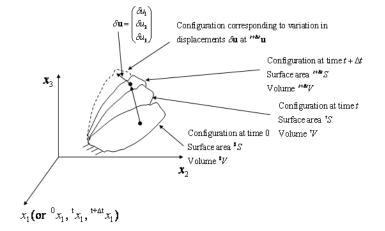




The Lagrangian formulation

We express equilibrium of the body at time  $t+\Delta t$  using the principle of virtual displacements

$$\int_{t+\Delta t_V} t+\Delta t \tau \mathcal{S}_{t+\Delta t} e_{ij} d^{t+\Delta t} V = t+\Delta t R$$



 $t^{t+\Delta t}\tau$ : Cartesian components of the Cauchy stress tensor

 $\delta_{t+\Delta t} e_{ij} = \frac{1}{2} \left( \frac{\partial \delta u_i}{\partial^{t+\Delta t} x_j} + \frac{\partial \delta u_j}{\partial^{t+\Delta t} x_i} \right) = \text{strain tensor corresponding to virtual displacements}$ 

 $\delta u_i$ : Components of virtual displacement vector imposed at time  $t + \Delta t$ 

 $t^{t+\Delta t}x_i$ : Cartesian coordinate at time  $t + \Delta t$ 

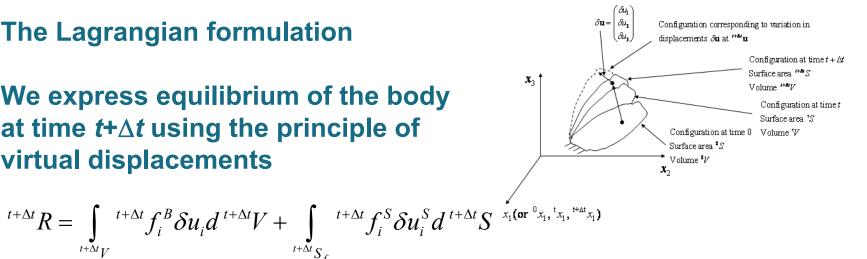
 $^{t+\Delta t}V$ : Volume at time  $t + \Delta t$ 

$${}^{t+\Delta t}R = \int_{t+\Delta t_V} {}^{t+\Delta t}f_i^B \delta u_i d^{t+\Delta t}V + \int_{t+\Delta t_{S_f}} {}^{t+\Delta t}f_i^S \delta u_i^S d^{t+\Delta t}S$$



The Lagrangian formulation

We express equilibrium of the body at time  $t+\Delta t$  using the principle of virtual displacements



where

 $f_{i}^{B}$ : externally applied forces per unit volume  $f_{i}^{S}$ : externally applied surface tractions per unit surface  $^{t+\Delta t}S_{f}$ : surface at time  $t + \Delta t$  $\delta u_i^S$ :  $\delta u_i$  evaluated at the surface  ${}^{t+\Delta t}S_f$ 

• The Lagrangian formulation

We recognize that our derivations from linear finite element theory are unchanged – but applied to the body in the configuration at time  $t+\Delta t$ 

#### In the further we introduce an appropriate notation:

Coordinates and displacements are related as:

$${}^{t}x_{i} = {}^{0}x_{i} + {}^{t}u_{i}$$
$${}^{t+\Delta t}x_{i} = {}^{0}x_{i} + {}^{t+\Delta t}u_{i}$$

Increments in displacements are related as:

$$_{t}u_{i}={}^{t+\Delta t}u_{i}-{}^{t}u_{i}$$

Reference configurations are indexed as e.g.:

 ${}^{t+\Delta t}_{0}f_{i}^{S}$  where the lower left index indicates the reference configuration

$$\tau_{t+\Delta t} \tau_{ij} = \tau_{t+\Delta t}^{t+\Delta t} \tau_{ij}$$

Differentiation is indexed as:

$${}^{t+\Delta t}_{0}u_{i,j} = \frac{\partial^{t+\Delta t}u_i}{\partial^{0}x_j}, \qquad {}^{0}_{t+\Delta t}x_{m,n} = \frac{\partial^{0}x_m}{\partial^{t+\Delta t}x_n}$$

## Aim of the present lecture

We have already formulated the continuum mechanich incremental equations of motion

$$\int_{t+\Delta t_V} t+\Delta t \tau \delta_{t+\Delta t} e_{ij} d^{t+\Delta t} V = t+\Delta t R$$

#### and

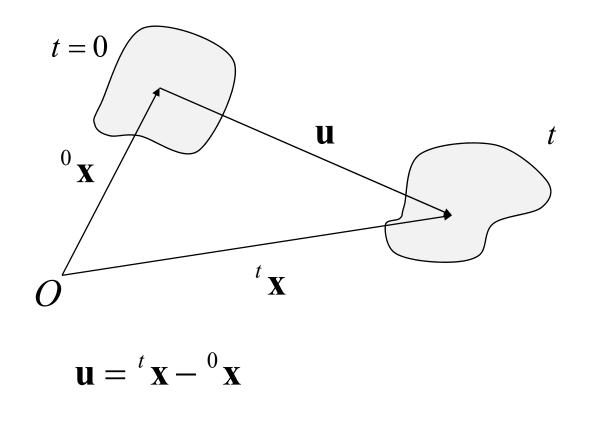
$${}^{t+\Delta t}R = \int_{t+\Delta t_V} {}^{t+\Delta t}f_i^B \delta u_i d^{t+\Delta t}V + \int_{t+\Delta t_{S_f}} {}^{t+\Delta t}f_i^S \delta u_i^S d^{t+\Delta t}S$$

a basic problem is that we dont know the configuration at time  $t+\Delta t$  (in linear analysis we always used the original configuration as basis)

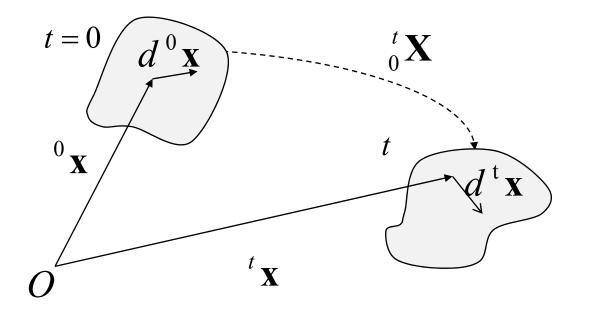
what we need to do now is to introduce appropriate stress and strain measures as well as constitutive relations

- As mentioned we must try to establish a description of the volume we consider such that we can express the internal virtual work in terms of an integral over a volume we know!
- Further we would like to be able to decompose the stresses and strains in an efficient manner keeping track of how the volume stretches and how it rotates (rigidly).

#### We consider a body under deformation at times 0 and t



We now consider the change of an infinitesimal gradient vector



The we can write  $d^{t}\mathbf{X} = {}^{t}\mathbf{X}({}^{0}\mathbf{X} + d^{0}\mathbf{X}, t) - {}^{t}\mathbf{X}({}^{0}\mathbf{X}, t)$ which is linear in the gradient why we have

$$d^{t}\mathbf{x} = {}_{0}^{t}\mathbf{X}d^{0}\mathbf{x}$$

#### • We can write the deformation gradient as

 ${}_{0}^{t}\mathbf{X} = \begin{bmatrix} \frac{\partial^{t} x_{1}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{1}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{1}}{\partial^{0} x_{3}} \\ \frac{\partial^{t} x_{2}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{2}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{2}}{\partial^{0} x_{3}} \\ \frac{\partial^{t} x_{3}}{\partial^{0} x_{1}} & \frac{\partial^{t} x_{3}}{\partial^{0} x_{2}} & \frac{\partial^{t} x_{3}}{\partial^{0} x_{3}} \end{bmatrix}$ The deformation gradient describes the stretches and rotations that the material fibers have undergone from time zero to time t

$${}_{0}^{t}\mathbf{X} = ({}_{0}\nabla^{t}\mathbf{x}^{T})^{T}, \text{ where } {}_{0}\nabla = \begin{bmatrix} \frac{\partial}{\partial^{0}x_{1}} \\ \frac{\partial}{\partial^{0}x_{2}} \\ \frac{\partial}{\partial^{0}x_{3}} \end{bmatrix}; \text{ and } {}^{t}\mathbf{x}^{T} = \begin{bmatrix} {}^{t}x_{1} & {}^{t}x_{2} & {}^{t}x_{3} \end{bmatrix}$$

it can be show that 
$${}_{0}^{t}\mathbf{X} = \left({}_{t}^{0}\mathbf{X}\right)^{-1}$$
 and  ${}^{t}\rho = \frac{{}^{0}\rho}{\det\left({}_{t}^{0}\mathbf{X}\right)}$ 



 $\frac{\partial^t x_1}{\partial^0 x_1} \quad \frac{\partial^t x_1}{\partial^0 x_2} \quad \frac{\partial^t x_1}{\partial^0 x_3}$ 

 $\frac{\partial^t x_3}{\partial^0 x_1} \quad \frac{\partial^t x_3}{\partial^0 x_2} \quad \frac{\partial^t x_3}{\partial^0 x_3}$ 

 $_{0}^{t}\mathbf{X} = (_{0}\nabla^{t}\mathbf{x}^{T})^{T}$ , where  $_{0}\nabla =$ 

 $\frac{\partial}{\partial^0 x_2}$ 

and  ${}^{t}\mathbf{x}^{T} = \left| {}^{t}x_{1} \right|$ 

 ${}_{0}^{t}\mathbf{X} = \begin{vmatrix} \frac{\partial^{t}x_{2}}{\partial^{0}x_{1}} & \frac{\partial^{t}x_{2}}{\partial^{0}x_{2}} & \frac{\partial^{t}x_{2}}{\partial^{0}x_{3}} \end{vmatrix}$ 

• Then we introduce the Cauchy-Green deformation tensor

The deformation gradient is also used to measure the stretch of a material fiber and the change in angle between fibers due to the deformation

$$_{0}^{t}\mathbf{C} = _{0}^{t}\mathbf{X}^{T} _{0}^{t}\mathbf{X}$$
 "Right Cauchy-Green deformation tensor"

$${}_{0}^{t}\mathbf{B} = {}_{0}^{t}\mathbf{X}_{0}^{t}\mathbf{X}^{T}$$
 "Left Cauchy-Green deformation tensor"



The deformation gradient

The deformation gradient can be decomposed into a unique product of two matrices

$${}_{0}^{t}\mathbf{X} = {}_{0}^{t}\mathbf{R} {}_{0}^{t}\mathbf{U}$$

 $_{0}^{t}\mathbf{U}$  Symmetric stretch matrix

<sup>t</sup><sub>0</sub>**R** Orthogonal rotation matrix

Referred to as a polar decomposition (illustrated in Ex 6.8)

Sometimes the indexes referring to time are omitted!

$$\mathbf{F}_{0}^{T} \mathbf{X} = \begin{bmatrix} \frac{\partial^{T} x_{1}}{\partial^{0} x_{1}} & \frac{\partial^{T} x_{1}}{\partial^{0} x_{2}} & \frac{\partial^{T} x_{2}}{\partial^{0} x_{3}} \\ \frac{\partial^{T} x_{2}}{\partial^{0} x_{1}} & \frac{\partial^{T} x_{2}}{\partial^{0} x_{2}} & \frac{\partial^{T} x_{2}}{\partial^{0} x_{3}} \\ \frac{\partial^{T} x_{3}}{\partial^{0} x_{1}} & \frac{\partial^{T} x_{3}}{\partial^{0} x_{2}} & \frac{\partial^{T} x_{3}}{\partial^{0} x_{3}} \end{bmatrix}$$

$$\mathbf{X} = \begin{pmatrix} \mathbf{0} \nabla^{T} \mathbf{x}^{T} \end{pmatrix}^{T}, \text{ where } \quad \mathbf{0} \nabla = \begin{bmatrix} \frac{\partial}{\partial^{0} x_{1}} \\ \frac{\partial}{\partial^{0} x_{2}} \\ \frac{\partial}{\partial^{0} x_{3}} \end{bmatrix}; \quad \text{and } \quad {}^{T} \mathbf{x}^{T} = \begin{bmatrix} {}^{T} x_{1} & {}^{T} x_{2} & {}^{T} x_{3} \end{bmatrix}$$

Decomposition of the deformation gradient

We continue by rewriting the deformation gradient

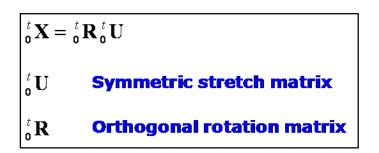
 $\mathbf{X} = \mathbf{R}\mathbf{U} = \mathbf{R}\mathbf{U}\mathbf{R}^T\mathbf{R} = \mathbf{V}\mathbf{R}$ 

U: right stretch matrix V: left stretch matrix

Further it can be shown (Ex 6.8) that :

 $\mathbf{U} = \mathbf{R}_L \mathbf{\Lambda} \mathbf{R}_L^T$ 

- $\Lambda$ : Principal stretches
- $\mathbf{R}_L$ : Direction of principal stretches





• Decomposition of the deformation gradient

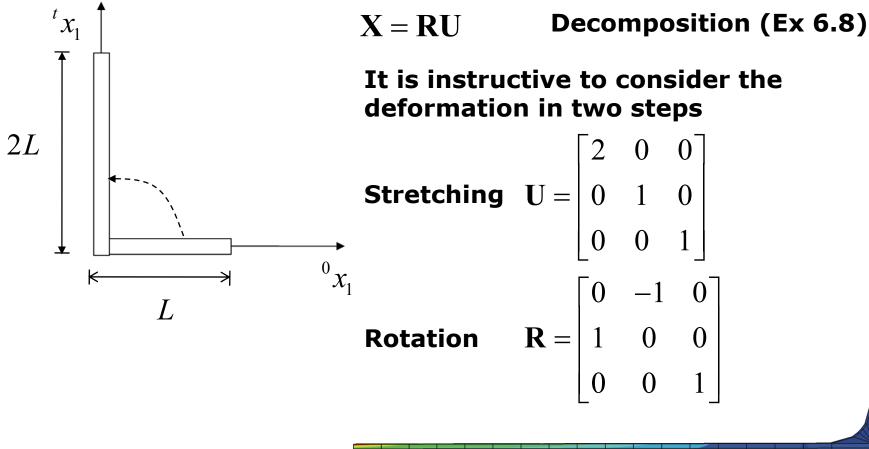
There is also:

 $\mathbf{U} = \mathbf{R}_L \mathbf{\Lambda} \mathbf{R}_L^T$ 

- $\Lambda$ : Principal stretches
- $\mathbf{R}_L$ : Direction of principal stretches

- $\mathbf{V} = \mathbf{R}_E \mathbf{\Lambda} \mathbf{R}_E^T$
- $\mathbf{R}_{E}$ : Base vectors of principal stretches in the stationary coordinate system

We consider a bar under stretch and rotation





• Using the decomposition of the deformation gradient we may rewrite the right and left Cauchy-Green deformation tensors:

The right Cauchy-Green deformation tensor:  $\mathbf{C} = \mathbf{X}^T \mathbf{X} = \mathbf{U} \mathbf{R}^T \mathbf{R} \mathbf{U} = \mathbf{U}^2$ 

The left Cauchy-Green deformation tensor:  $\mathbf{B} = \mathbf{X}\mathbf{X}^T = \mathbf{V}\mathbf{R}\mathbf{R}^T\mathbf{V} = \mathbf{V}^2$ 



• We now proceed from deformations to strains ③

The strain may be understood as the stretch per unit length why we can assess the strain through the inner product between two infinitesimal vectors before and after deformation

$$d^{t}\mathbf{x}_{1} \cdot d^{t}\mathbf{x}_{2} - d^{0}\mathbf{x}_{1} \cdot d^{0}\mathbf{x}_{2} = (\mathbf{X}d^{0}\mathbf{x}_{1}) \cdot (\mathbf{X}d^{0}\mathbf{x}_{2}) - d^{0}\mathbf{x}_{1} \cdot d^{0}\mathbf{x}_{2}$$
$$= d^{0}\mathbf{x}_{1} \cdot (\mathbf{C} - \mathbf{I}) \cdot d^{0}\mathbf{x}_{2}$$
Green-Lagrange strain:  $\frac{1}{2}(\mathbf{C} - \mathbf{I})$ 

$$d^{t} \mathbf{x}_{1} \bullet d^{t} \mathbf{x}_{2} - d^{0} \mathbf{x}_{1} \bullet d^{0} \mathbf{x}_{2} = d^{t} \mathbf{x}_{1} \bullet d^{t} \mathbf{x}_{2} - (\mathbf{X}^{-1} d^{t} \mathbf{x}_{1}) \bullet (\mathbf{X}^{-1} d^{t} \mathbf{x}_{2})$$
$$= d^{t} \mathbf{x}_{1} \bullet (\mathbf{I} - \mathbf{B}^{-1}) \bullet d^{t} \mathbf{x}_{2}$$
  
Almansi strain:  $\frac{1}{2} (\mathbf{I} - \mathbf{B}^{-1})$ 



Lets see an example (one-dimensional) We assume the following deformation gradient matrix

$$\mathbf{X} = \begin{bmatrix} \frac{l}{L} & 0 & 0\\ 0 & 1 & 0\\ 0 & 0 & 1 \end{bmatrix}; \quad \text{i.e. pure stretch}$$

$$d^{t} \mathbf{x}_{1} \cdot d^{t} \mathbf{x}_{1} - d^{0} \mathbf{x}_{1} \cdot d^{0} \mathbf{x}_{1} = (\frac{l}{L} d^{0} \mathbf{x}_{1}) \cdot (\frac{l}{L} d^{0} \mathbf{x}_{1}) - d^{0} \mathbf{x}_{1} \cdot d^{0} \mathbf{x}_{1}$$

$$= d^{0} \mathbf{x}_{1} \cdot (\frac{l^{2}}{L^{2}} - 1) \cdot d^{0} \mathbf{x}_{1}$$
or equivalently
$$= d^{0} \mathbf{x}_{1} \cdot d^{0} \mathbf{x}_{1} - (\mathbf{X}^{-1} d^{t} \mathbf{x}_{1}) \cdot (\mathbf{X}^{-1} d^{t} \mathbf{x}_{1})$$

$$= d^{t} \mathbf{x}_{1} \cdot (1 - \frac{L^{2}}{l^{2}}) \cdot d^{t} \mathbf{x}_{1}$$
of Finite Elements II

Method c

Lets see an example (one-dimensional) •

Green-Lagrange strains:

Almansi strains:

$$\mathbf{E} = \frac{1}{2} \left( \frac{l^2}{L^2} - 1 \right)$$
$$\mathbf{A} = \frac{1}{2} \left( 1 - \frac{L^2}{l^2} \right)$$

1 72

for infinitesimal strains there is:

$$\frac{1}{2}\left(\frac{l^2}{L^2} - 1\right) = \frac{1}{2}\frac{(u+L)^2 - L^2}{L^2} \approx \frac{u}{L}$$
$$\frac{1}{2}\left(1 - \frac{L^2}{l^2}\right) = \frac{1}{2}\frac{(u+L)^2 - L^2}{l^2} \approx \frac{u}{l} \approx \frac{u}{L}$$

11

and

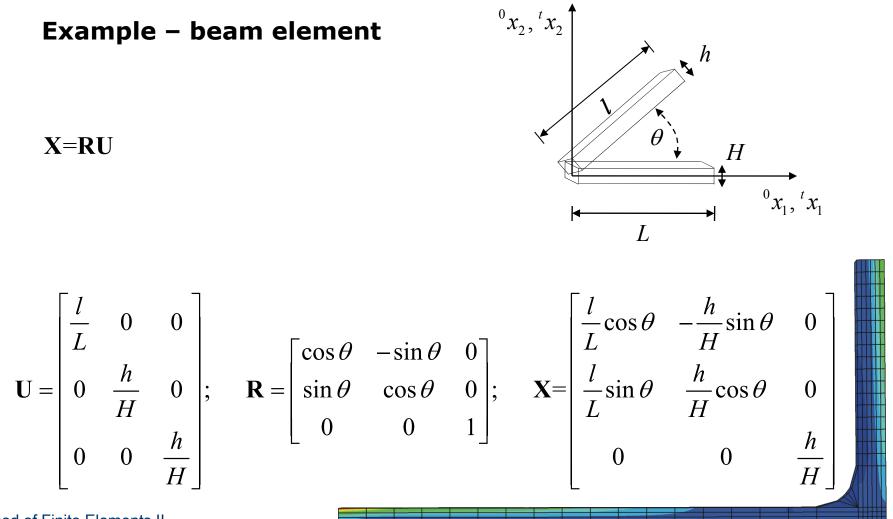
• We now consider the tensor components of the strain tensors

#### **Green-Lagrange strains**

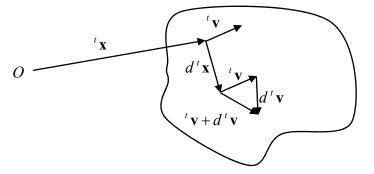
$$\mathbf{\varepsilon} = \varepsilon_{ij} \mathbf{e}_i \otimes \mathbf{e}_j = \frac{1}{2} \left\{ \frac{\partial u_i}{\partial^o x_j} + \frac{\partial u_j}{\partial^o x_i} + \frac{\partial u_k}{\partial^o x_i} \frac{\partial u_k}{\partial^o x_i} \right\} \mathbf{e}_i \otimes \mathbf{e}_j$$

#### Almansi strains

$$\boldsymbol{\alpha} = \boldsymbol{\alpha}_{ij} \mathbf{e}_i \otimes \mathbf{e}_j = \frac{1}{2} \left\{ \frac{\partial u_i}{\partial^t x_j} + \frac{\partial u_j}{\partial^t x_i} + \frac{\partial u_k}{\partial^t x_i} \frac{\partial u_k}{\partial^t x_i} \right\} \mathbf{e}_i \otimes \mathbf{e}_j$$



Now we consider the velocity gradient tensor – the difference in velocity of two points infinitesimally close



We can write change of velocity over space as a linear function of the distance in space

 $d^{t}\mathbf{v} = \mathbf{L}d^{t}\mathbf{x}$ 

where L is given through the gradient of the velocity field at time t

 $L = v \otimes \nabla_x$  This is the velocity gradient tensor  $\odot$ 





 $d^{t}\mathbf{x} = \mathbf{X}d^{0}\mathbf{x}$ 

#### which leads us to:

$$d^{t} \mathbf{v} = \dot{\mathbf{X}} d^{0} \mathbf{x}$$

$$\Downarrow$$

$$d^{t} \mathbf{v} = \mathbf{L} \mathbf{X} d^{0} \mathbf{x}$$

$$\Downarrow$$

$$\mathbf{L} = \dot{\mathbf{X}} \mathbf{X}^{-1}$$

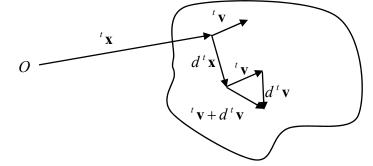
 $\mathbf{L} = \mathbf{D} + \mathbf{W}$  decomposition

$$\mathbf{D} = \frac{1}{2} (\mathbf{L} + \mathbf{L}^T) = \frac{1}{2} (\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i}) \mathbf{e}_i \otimes \mathbf{e}_j$$

deformation rate tensor

$$\mathbf{W} = \frac{1}{2} (\mathbf{L} - \mathbf{L}^T) = \frac{1}{2} (\frac{\partial v_i}{\partial x_i} - \frac{\partial v_j}{\partial x_i}) \mathbf{e}_i \otimes \mathbf{e}_j$$

spin/rotation rate tensor



 $t \mathbf{x}$ 

 $d^{t}$ 

And then we may derive the Green-Lagrange velocity strain tensor

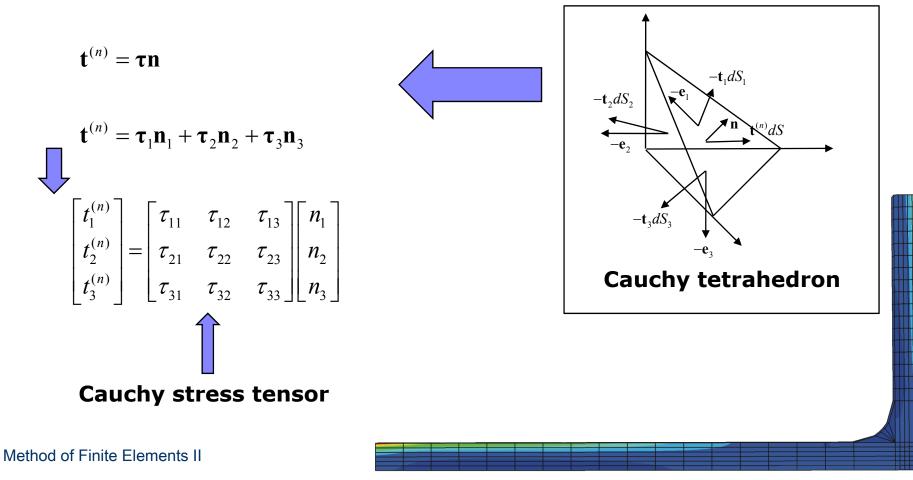
 $\dot{\boldsymbol{\varepsilon}} = {}_{0}^{t} \mathbf{X}^{T} \mathbf{D}_{0}^{t} \mathbf{X} \qquad \mathbf{D} = {}_{t}^{0} \mathbf{X}^{T} \dot{\boldsymbol{\varepsilon}}_{t}^{0} \mathbf{X}$ 

We could also just have differentiated the Green-Lagrange strain tensor with resect to time

$$\dot{\boldsymbol{\varepsilon}} = \frac{1}{2} \begin{pmatrix} t & \dot{\mathbf{X}}^T & t \\ 0 & \dot{\mathbf{X}}^T & 0 \end{pmatrix} \mathbf{X} + \begin{pmatrix} t & \mathbf{X}^T & t \\ 0 & \dot{\mathbf{X}}^T & 0 \end{pmatrix}$$

#### Finally we need to establish the stresses

We start by introducing the Cauchy stresses:



**Finally we introduce the second Piola-Kirchoff stresses:** 

$${}_{0}^{t}\mathbf{S} = \frac{{}^{0}\boldsymbol{\rho}}{{}^{t}\boldsymbol{\rho}} {}^{0}_{t}\mathbf{X}^{t}\boldsymbol{\tau}{}^{0}_{t}\mathbf{X}^{T}$$

these are so-called work conjugate to the Green-Lagrange strains

**Rigid body motions do not induce strains/stresses** 

the strain and stress tensors are invariant in regard to rotations

Worthwhile to consult Ex 6.14-6.15 ©

#### We remember that we set out to solve the following equation:

$$\int_{t+\Delta t_V} t^{t+\Delta t} \tau \delta_{t+\Delta t} e_{ij} d^{t+\Delta t} V = t^{t+\Delta t} R$$

 $t^{t+\Delta t}\tau$ : Cartesian components of the Cauchy stress tensor

 $\delta_{t+\Delta t} e_{ij} = \frac{1}{2} \left( \frac{\partial \delta u_i}{\partial^{t+\Delta t} x_j} + \frac{\partial \delta u_j}{\partial^{t+\Delta t} x_i} \right) = \text{strain tensor corresponding to virtual displacements}$ 

 $\delta u_i$ : Components of virtual displacement vector imposed at time  $t + \Delta t$ 

 $x_{i}^{t+\Delta t}$ : Cartesian coordinate at time  $t + \Delta t$ 

 $^{t+\Delta t}V$ : Volume at time  $t + \Delta t$ 

$${}^{t+\Delta t}R = \int_{t+\Delta t_V} {}^{t+\Delta t}f_i^B \delta u_i d^{t+\Delta t}V + \int_{t+\Delta t_{S_f}} {}^{t+\Delta t}f_i^S \delta u_i^S d^{t+\Delta t}S$$

#### We remember that we set out to solve the following equation:

$$\int_{t+\Delta t_{V}} t^{t+\Delta t} \tau \mathcal{S}_{t+\Delta t} e_{ij} d^{t+\Delta t} V = t^{t+\Delta t} R$$

#### Two schemes have been formulated for this namely:

#### The Total Lagrangian (TL) formulation

$$\int_{0_V} \int_{0_V} \int_{0}^{t+\Delta t} \mathcal{E}_{ij} d^0 V = \int_{0}^{t+\Delta t} \mathcal{R}$$

The Updated Lagrangian (UL) formulation

$$\int_{V} \int_{V} \int_{V} \delta_{ij} \delta_{t+\Delta t} \varepsilon_{ij} d^{t} V = \int_{V} \delta_{ij} \delta_{t+\Delta t} \delta_{ij} \delta_{ij} \delta_{ij} \delta_{ij} \delta_{t+\Delta t} \delta_{ij} \delta_{ij}$$

The resulting equations of motion for time *t* may be derived to:

The Total Lagrangian (TL) formulation

$$\int_{O_{V}} {}_{0}C_{ijrs\ 0}e_{rs}\delta_{0}e_{ij}d^{0}V + \int_{O_{V}} {}_{0}{}^{t}S_{ij}\delta_{0}\eta_{ij}d^{0}V = {}^{t+\Delta t}R - \int_{O_{V}} {}_{0}{}^{t}S_{ij}\delta_{0}e_{ij}d^{0}V$$

The Updated Lagrangian (UL) formulation

$$\int_{t_V} {}_{_{0}}C_{ijrs\ t}e_{rs}\delta_te_{ij}d^tV + \int_{_{0_V}} {}^{_{t}}\tau_{ij}\delta_t\eta_{ij}d^tV = {}^{_{t+\Delta t}}R - \int_{_{t_V}} {}^{_{t}}\tau_{ij}\delta_te_{ij}d^tV$$

Finally – in practice it is often sufficient to account for only material non-linearity

# In this case the TL and the UL formulations become identical.